

ENGLISH

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SYSTEM

At a glance

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# CENTAX-KS AT A GLANCE

A coupling for diesel-driven compressor or pump drives should have one primary feature: to be as uncompromising regarding ease in mounting as our new CENTAX-KS.

The torsionally soft coupling is based on a rubber element that features high torsional flexibility and radial movability vulcanised to a glass-fibre reinforced plastic flange with an integrated hub on the output side. The coupling is designed as a single unit and can be plugged in axially to ensure maximum ease in mounting. And, the design is also impressive considering its dimensions and cost.

The torsional flexibility of the CENTAX-KS dampens torsional vibrations and impacts and compensates for any axial and radial misalignments due to operation. The rubber element for this coupling is available in various designs. Currently, the series covers torques ranging from 400 to 800 Nm.

#### **Features**

High torsional flexibility
Compact dimensions
Maximum ease in mounting
Low-cost

#### Areas of application

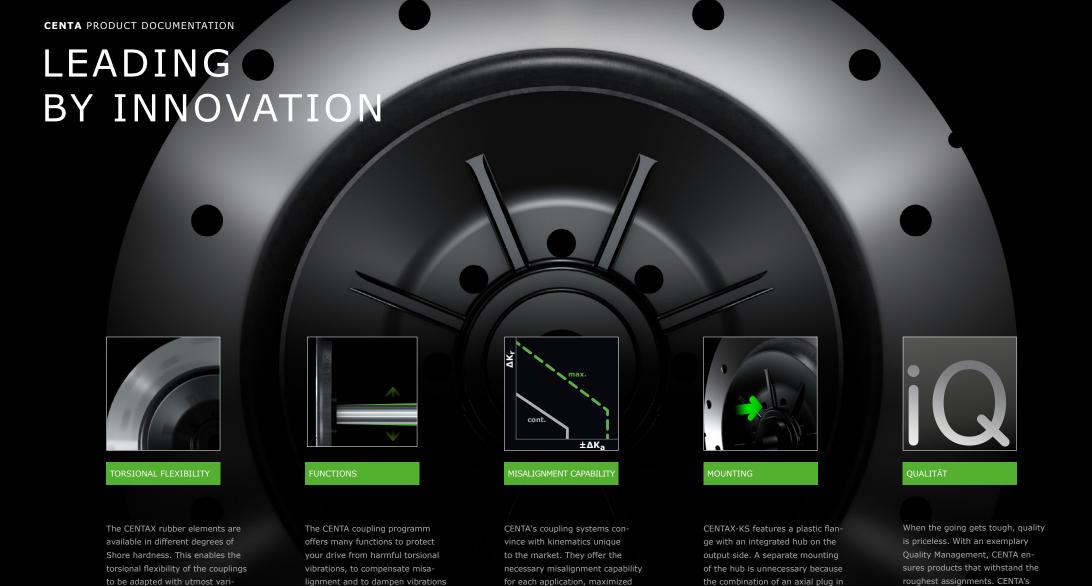


Compressor drives
Pump drives
Hydrostatic drives

#### Torque range

0,4 - 0,8 kNm Higher torques upon request

For efficient torque transmission



for each application, maximized

endurance tests (type test).

ping of vibration and noise.

through numeric calculations and

Ensuring a reliable compensation

of misalignments even with short drive shafts and an efficient dam-

lignment and to dampen vibrations

and noise. Reliable power trans-

mission for your applications with

an optimum of features.

ability to the specific application.

Torsional vibrations and impacts

are reliably dampened.

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of the coupling provides maximum

ease in mounting.

coupling systems are more than

entertains the vision of intelligent products that meet the highest

requirements in terms of design

and quality.

the sum of their parts. CENTA

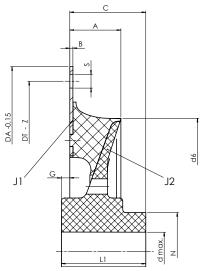
# TECHNICAL DATA

Questions on product selection?
We will gladly assist → www.centa.info/contact



TECHNICAL DATA			↓ SIZES 22-28										
1	2	3	4	5	6*	7*	8	9	10	12	13	14	
Size	Rubber quality*	Nominal torque	Maximum torque	Continuous vibratory torque	Permissible power loss	Dynamic torsional stiffness	Relative damping	Speed	Permissible axial displacement	Permissible radial displacement	Radial stiffness	Permissible angular displacement	Flange size
	[Shore A]	TĸN	T <sub>Kmax</sub>	Tĸw	Рку	C <sub>Tdyn</sub>	Ψ	n <sub>max</sub>	ΔKa	$\Delta K_r$	Cr	ΔKw	SAE J620
		[kNm]	[kNm]	[kNm]	[W]	[kNm/rad]		[min <sup>-1</sup> ]	[mm]	[mm]	[kN/mm]	[°]	
22	50	0,40	0,80	0,16	100	2,6	1,05	5000	±2	0,6	0,75	0,5	7,5; 8; 10
22	60	0,50	1,00	0,20	100	3,2	1,10				0,85	0,5	
25	50	0,55	1,10	0,22	110	3,5	1,05	5000	±2	0,6	1,00	0,5	0.10
25	60	0,68	1,36	0,27		4,3	1,10				1,10	0,5	8; 10
20	50	0,63	1,26	0,25	120	4,0	1,05			1,10	0,5	10: 11 5	
28	60	0,80	1,60	0,32	120	5,0	1,10	5000	±2	0,6	1,20	0,5	10; 11,5

<sup>\*</sup> preliminary values



DIMENSIONS			SIZES	22-28															
Size	Nominal torque	Flange size					Dimensions						Mass moments of intertia and masses						
	T <sub>KN</sub> [kNm]	SAE J620	Da	Dτ	Z	S	d₅	L <sub>1</sub>	N	d <sub>max</sub>	A	В	С	G	J₁ [kg∙	J <sub>2</sub> cm²]	m <sub>1</sub>	m₂ [kg]	Mtotal
		7,5	241,3	222,3	8x45°	9	174	55	51	40	33,3	2	49,5	5,5	0,006	0,002	0,71	0,56	1,27
22	0,40 - 0,50	8	263,5	244,5	6x60°	11	174	55	51	40	33,3	2	49,5	5,5	0,008	0,002	0,84	0,56	1,40
	10	314,3	295,3	8x45°	11	174	55	51	40	33,3	2	49,5	5,5	0,015	0,002	1,19	0,56	1,75	
25	25 25 262	8	263,5	244,5	6x60°	11	187	55	51	40	35,4	2	49,5	5,5	0,009	0,003	0,91	0,65	1,56
25 0,55 - 0,68	10	314,3	295,3	8x45°	11	187	55	51	40	35,4	2	49,5	5,5	0,016	0,003	1,26	0,65	1,91	
28 0,63 - 0,80	0.63 0.00	10	314,3	295,3	8x45°	11	194	55	51	40	36,6	2	49,5	5,5	0,016	0,003	1,25	0,76	2,01
	11,5	352,4	333,4	8x45°	11	194	55	51	40	36,6	2	49,5	5,5	0,025	0,003	1,56	0,76	2,32	

# **CENTAX**-KS APPENDIX

#### EXPLANATION OF THE TECHNICAL DATA

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#### EXPLANATION OF THE TECHNICAL DATA



This spontaneously selected figure designates the size of the coupling.

#### 2 Rubber quality Shore A

This figure indicates the nominal shore hardness of the elastic element.

The nominal value and the effective value may deviate within given tolerance ranges.

#### 3 Nominal torque TĸŊ [kŊm]

Average torque which can be transmitted continuously over the entire speed range.

# 4 Maximum torque [kNm] This is the torque that may

This is the torque that may occur occasionally and for a short period up to 1.000 times and may not lead to a substantial temperature rise in the rubber element.

In addition the following maximum torques may occur:

$\Delta T_{\text{Kmax}} = 1,8  \text{x} T_{\text{KN}}$	Peak torque range (peak to peak) between maximum and minimum torque, e.g. switch- ing operation.
T <sub>Kmax1</sub> = 1,5 x T <sub>KN</sub>	Temporary peak torque (e.g. passing through resonances). $\Delta T_{\text{Kmax}}$ or $T_{\text{Kmax}1}$ may occur 50.000 times alternating or 100.000 times swelling.
T <sub>Kmax2</sub> = 4,5 x T <sub>KN</sub>	Transient torque rating for very rare, extraordinary conditions (e.g. short circuits)

# 5 Continuous vibratory torque

1,0

0,8

0,6

0,4

0,2

# Tkw [kNm] Amplitude of the continuously permis-

20

30

50

60

sible periodic torque fluctuation with a basic load up to the value  $T_{KN}$ .

The frequency of the amplitude has no influence on the permissible continuous vibratory torque. Its main influence on the coupling temperature is taken into consideration in the calculation of the power loss.

# Operating torque T<sub>Bmax</sub> [kNm]

The maximum operating torque results of  $T_{\mbox{\scriptsize KN}}$  and  $T_{\mbox{\tiny KW}}$ 

# 6 Permissible Power Loss P<sub>KV</sub> [kW]

70

Damping of vibrations and displacement results in power loss within the rubber element.

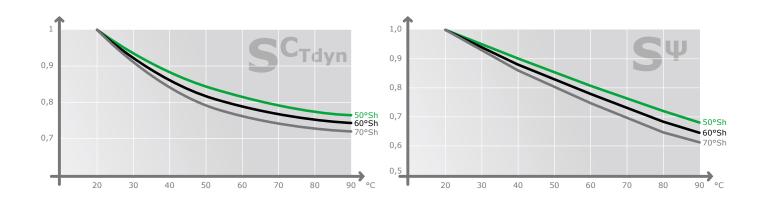
The permissible power loss is the maximum heat (converted damping work into heat), which the rubber element can dissipate continuously to the environment without the maximum allowable temperature being exceeded.

The given permissible power loss refers to an ambient temperature of 30° C.

If the coupling is to be operated at higher ambient temperatures, temperature factor  $S^{P_{KV}}$  has to be taken into consideration in the calculation.

The coupling can momentarily withstand an increase of the permissible power loss for a short period under certain operation modes (e.g. misfiring). CENTA keeps record of exact parameters.

#### EXPLANATION OF THE TECHNICAL DATA



# Dynamic torsional stiffness C<sub>Tdyn</sub> [kNm/rad]

The dynamic torsional stiffness is the relationship of the torque to the torsional angle under dynamic loading.

The torsional stiffness may be linear or progressive depending on the coupling design and material.

The value given for couplings with linear torsional stiffness considers following terms:

- 50% of Tkn pre-load
- 25% of Tkn vibratory torque (amplitude)
- 20°C ambient temperature
- 10 Hz frequency

For couplings with progressive torsional stiffness only the pre-load value changes as stated.

The tolerance of the torsional stiffness is  $\pm 15\%$  if not stated otherwise.

The following influences need to be considered if the torsional stiffness is required for other operating modes:

- Temperature
  - $\label{thm:ligher temperature reduces the dynamic torsional stiffness. \\$
  - Temperature factor  $S^{\text{\scriptsize C}_{\text{\scriptsize Tdyn}}}$  has to be taken into consideration in the calculation.
- Frequency of vibration
  - Higher frequencies increase the torsional stiffness.
  - By experience the dynamic torsional stiffness is 30% higher than the static stiffness. CENTA keeps record of exact parameters.
- Amplitude of vibratory torque

Higher amplitudes reduce the torsional stiffness, therefore small amplitudes result in higher dynamic stiffness. CENTA keeps record of exact parameters.

#### Relative damping Ψ

The relative damping is the relationship of the damping work to the elastic deformation during a cycle of vibration.

8

The larger this value  $[\psi]$ , the lower is the increase of the continuous vibratory torque within or close to resonance.

The tolerance of the relative damping is  $\pm 20\%$ , if not otherwise stated.

The relative damping is reduced at higher temperatures.

Temperature factor  $S^{\boldsymbol{\psi}}$  has to be taken into consideration in the calculation.

The vibration amplitude and frequency only have marginal effect on the relative damping.

#### Speed [min<sup>-1</sup>]

Maximum speed of the coupling element, which may occur occasionally and for a short period (e.g. overspeed).

The characteristics of mounted parts may require a reduction of the maximum speed (e.g. outer diameter or material of brake discs).

The maximum permissible speed of highly flexible coupling elements is normally 90% thereof.

# 10 Permissible axial displacement [mm]

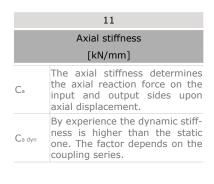
Continuous permissible axial displacement of the coupling.

This is the sum of displacement by assembly as well as static and dynamic displacements during operation.

Maximum axial displacement of the coupling, which may occur occasionally for a short period (e.g. extreme load).

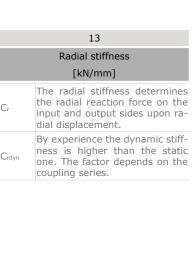
ΔK<sub>a max</sub> The concurrent occurrence of different kinds of displacements is handled in technical documents (displacement diagrams, data sheets, assembly instructions).

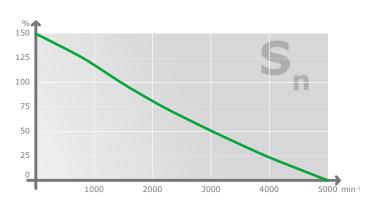
#### EXPLANATION OF THE TECHNICAL DATA



	12									
Permissible radial displacement										
[mm]										
ΔKr	Continuous permissible radial displacement of the coupling. This is the sum of displacement by assembly as well as static and dynamic displacements during operation.  The allowable radial displacement depends on the operation speed and may require adjustment (see diagrams of the coupling series).									
∆Kr max	Maximum radial displacement of the coupling, which may occur occasionally and for a short pe- riod without consideration of the operation speed (e.g. extreme overload). The concurrent occurrence of									
	different kinds of displacements is handled in technical documents (displacement diagrams, data sheets, assembly instructions).									

# adial J. Cr nent tatic duraceition ustcount of cpef the eme e of ents ocuams, rruc-





Continuous permissible angular displacement of the coupling.

This is the sum of displacement by assembly as well as static and dynamic displacements during operation.

 $\Delta K_w$ 

The allowable angular displacement depends on the operation speed and may require adjustment (see diagrams of the coupling series).

Maximum angular displacement of the coupling, which may occur occasionally and for a short period without consideration of the operation speed (e.g. extreme overload).

The concurrent occurrence of different kinds of displacements is handled in technical documents (displacement diagrams, data sheets, assembly instructions).

#### 15 Angular stiffness [kNm/°]

The angular stiffness determines the restoring bending moment on the input and output sides upon angular displacement.

By experience the dynamic stiffness is higher than the static one. The factor depends on the coupling series.

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Rev. CX-KS-EN-01-15

1. This catalog supersedes previous editions.

This catalog shows the extent of our CENTAX®-SEC coupling range at the time of printing. This program is still being extended with further sizes and series. Any changes due to technological progress are reserved.

We reserve the right to amend any dimensions or detail specified or illustrated in this publication without notice and without incurring any obligation to provide such modification to such couplings previously delivered. Please ask for an application drawing and current data before making a detailed coupling selection.

2. We would like to draw your attention to the need of preventing accidents or injury. No safety guards are included in our supply.

#### 3. TRADEMARKS

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#### 4. Torsional responsibility

The responsibility for ensuring the torsional vibration compatibility of the complete drive train, rests with the final assembler. As a component supplier CENTA is not responsible for such calculations, and cannot accept any liability for gear noise/-damage or coupling damage caused by torsional vibrations.

CENTA recommends that a torsional vibration analysis (TVA) is carried out on the complete drive train prior to start up of the machinery. In general torsional vibration analysis can be undertaken by engine manufacturers, consultants or classic fication societies. CENTA can assist with such calculations using broad experience in coupling applications and torsional vibration analysis.

- 5. Copyright to this technical dokument is held by CENTA Antriebe Kirschey GmbH.
- 6. The dimensions on the flywheel side of the couplings are based on the specifications given by the purchaser. The responsibility for ensuring dimensional compatibility rests with the assembler of the drive train. CENTA cannot accept liability for interference between the coupling and the flywheel or gearbox or for damage caused by such interference.
- 7. All technical data in this catalog are according to the metric SI system. All dimensions are in mm. All hub dimensions (N, N<sub>1</sub> and N<sub>2</sub>) may vary, depending on the required finished bore. All dimensions for masses (m), inertias (J) and centres of gravity (S) refer to the maximum bore diameter.



CENTA is the leading producer of flexible couplings for rail, industrial, marine and power generating applications. Worldwide.

#### **HEAD OFFICE**

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